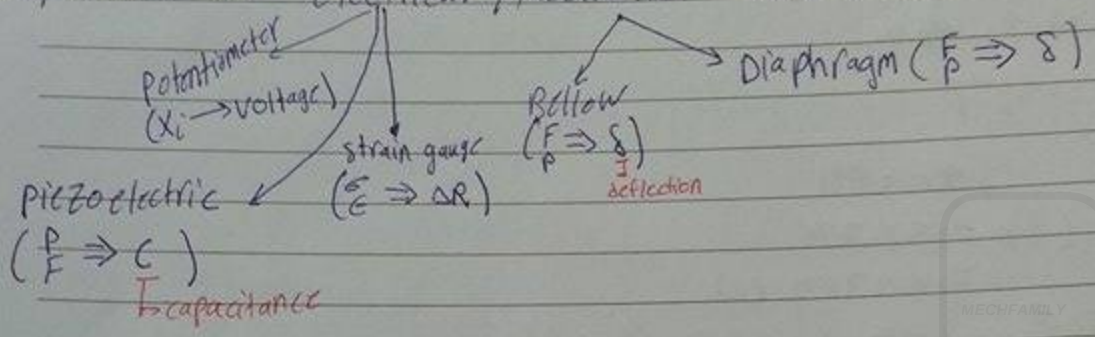


12 → 1:30
1 → 2:30
2 → 3:30

Hydrometer : measures viscosity
omega.com
engineersedge.com
etuniba.com

ch6) Basics of electrical/mechanical Transducers



RTD
Thermistor } $T_{em} \rightarrow \Delta R$

Capacitor $C = \epsilon \frac{A}{d}$ → overlap Area

strain Gauge xp

Beam is detector and primary transducer (detects signal)
strain gauge secondary transducer (force)

detectors only (thermometer)

Bourdon tube : pressure gauge
UDL : uniformly distributed load

$$K_s = \frac{dm}{P} = \frac{3(1-\nu^2) D^4}{256 E t^3}$$

Ex 5280

maximum shear stress at certain pressure

$$S_m < 0.5t$$

$t \uparrow, K_s \downarrow$

For this stress or pressure
if $S_m > 0.5t$ → thickness
→ thickness

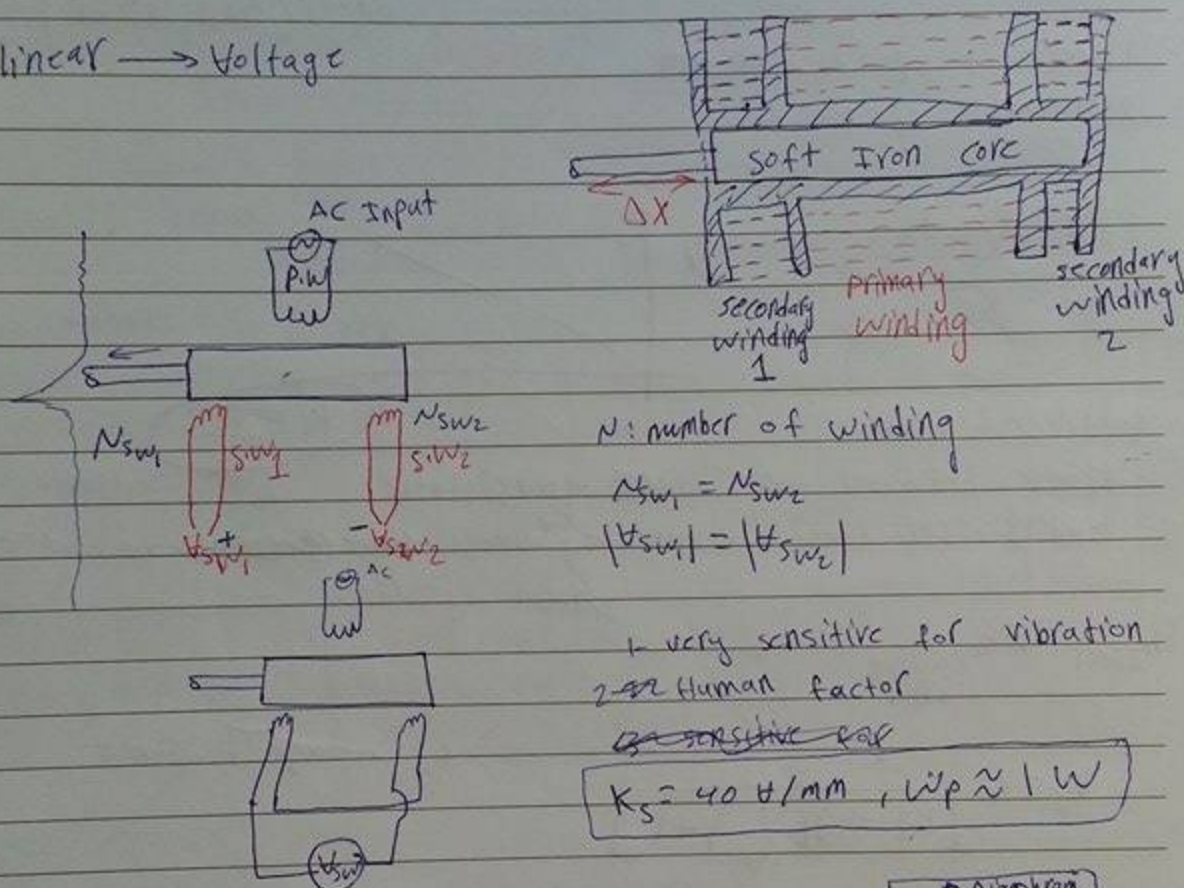
Electrical Transducer

1) Inductance (LVDT)

LVDT: Linear variable differential Transformer

small distance
(crack)

used for crack displacement measurement

linear $\rightarrow$ Voltage

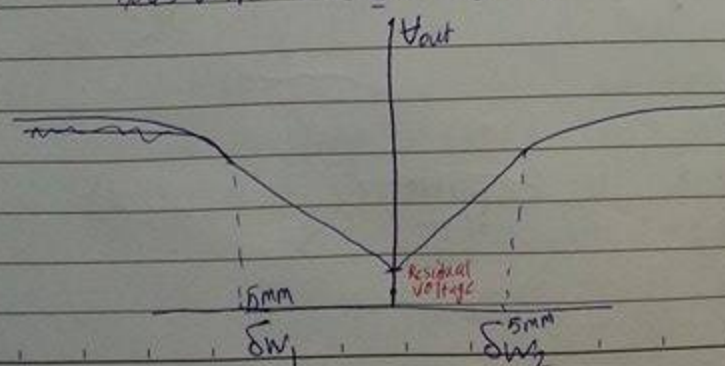
Iron: to concentrate the magnetic field

soft: to reduce the residual voltage

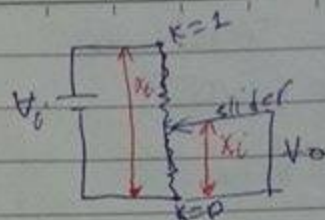
does not work as a transformer

$$V_{sw} = f(\delta_m)$$

$$= f(p)$$

Residual: ~~not a~~
Known error
due to Roughness

2) potentiometer

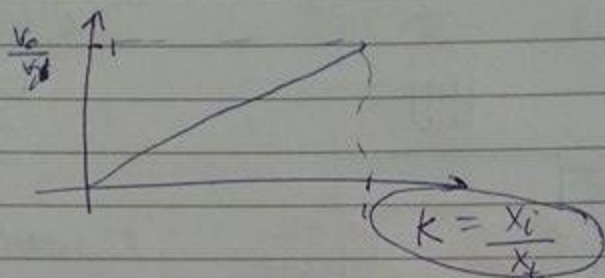


linear (translation)

linear : 2 to 500 mm

Rotational : 10° to 60°

$$V_0 = \frac{V_i}{K_s} \cdot X_i$$



source of errors:-

- 1- distance between wires cause quasistatic
- 2- loading error : as a result of voltage divider (internal resistance)

$$V_{0\text{Ideal}} > V_{0\text{Loading error}}$$

$$V_{0L} = V_i \left[\frac{K}{K(1-K) \left(\frac{R_p}{R_m} \right) + 1} \right]$$

R_m : voltage resistance

$$\lim_{R_m \rightarrow \infty} V_{0L} = V_{0I}$$

for good linearity :- $E \rightarrow 0$

$$I = R_p V$$

$$\text{OR } R_m \uparrow \uparrow$$

for $K \uparrow$:- $V_i \uparrow$ problem of power rating (مشكلة الطاقة)

$$P = \frac{V_c^2}{R_p}$$

solution for k^2 , nonlinearity $\downarrow$
 CerMet
 thin metal Film



Moulded carbon or moulded metal

Temperature transducers

- RTD

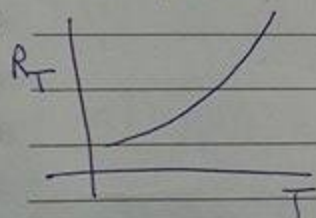
$$\int \Delta T \rightarrow \Delta R$$

- Thermistor

RTD

- metal

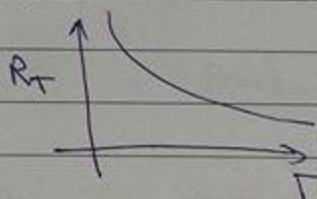
$R \uparrow$ as $T \uparrow$



thermistor

semi conductor

$R \downarrow$ as $T \uparrow$



Low temperature

High sensitivity at high
 temp so used
 for high temp
 measure.

melting temp $>$ temp working

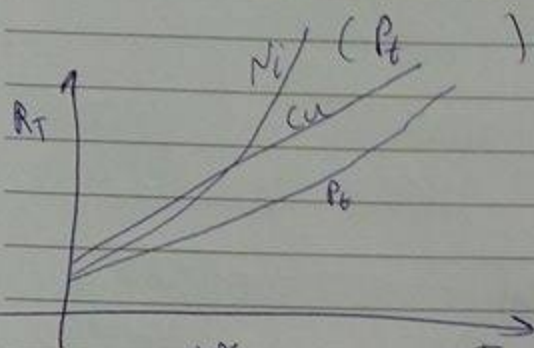
material 1) high melting point

Range is linear

R_T and T are linear
 chemical composition Homogeneous material
 geometrical (uniform geometrical)

corrosion resistance - 2
(plugged) non-toxic - 0

platinum RTD) حساس جدا



	α (slope of curve)	T_{min}	T_{max}	$T_{melting}$	
Pt	0.39	-260°C	1100°C	1773	
Cu	0.39	0	180	1083	
Ni	0.62	-220	300	1455	
Thiostone	0.45	-200	1000	3390	Highly Brittle

RTD equation

$$R_T = R_{T_0} \left[1 + \alpha_1 \Delta T + \alpha_2 \Delta T^2 + \dots \right]$$

T_0 : calibrated (reference) temperature

R_{T_0} : resistance at T_0

$$\Delta T = T - T_0$$

Thermistor

$$R_{T_1} = R_{T_2} \left(e^{\beta \left(\frac{1}{T_1} - \frac{1}{T_2} \right)} \right)$$

β : coeff. of variation between resistance and temp

~~RTD approximation~~

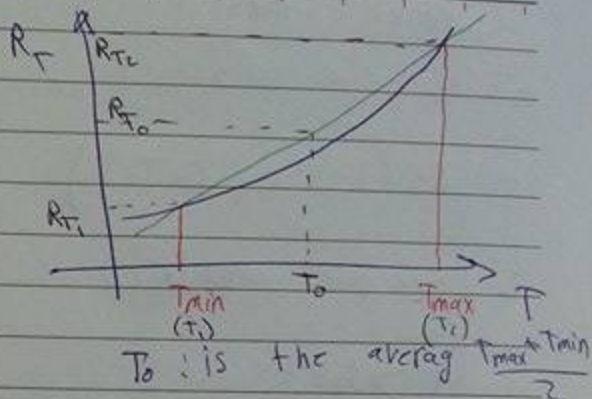
RTD $\rightarrow$ linear
 $\rightarrow$ quadratic

$$R_T = R_{T_0} [1 + \alpha_{T_0}(T - T_0)]$$

$$\alpha_{T_0} = \frac{1}{R_{T_0}} \left[\frac{\Delta R_T}{\Delta T} \right]$$

$\xrightarrow{R_{T_2} - R_{T_1}}$
 $\xleftarrow{T_2 - T_1}$

R_T : winston Bridge $\rightarrow$ 3 wires & 4 wires



$$\frac{R_2}{R_1} = \frac{RTD}{R_3}$$

quadratic

(I) $(T_1 \rightarrow T_0)$

(II) $(T_0 \rightarrow T_2)$

[I] $R_{T_1} = R_{T_0} [1 + \alpha_1(T_1 - T_0) + \alpha_2(T_1 - T_0)^2]$

[II] $R_{T_2} = R_{T_0} [1 + \alpha_1(T_2 - T_0) + \alpha_2(T_2 - T_0)^2]$

$$\frac{R_{T_1}}{R_{T_0}} + \frac{R_{T_2}}{R_{T_0}} = 2 + 2\alpha_2(T_2 - T_0)^2 \quad (\text{take five digits})$$

take average

$R_{T_0} \leftarrow 0^\circ\text{C}$

at $0^\circ\text{C} \leftarrow R_{T_0}$

at average temp $\leftarrow R_{T_0}$

take Range 1 dala

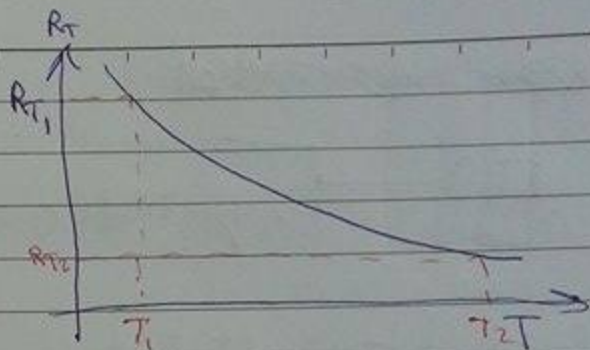
take Range 2 dala

take Range 3 dala

~~Thermistor~~ thermistor
 $\Delta T \leq 50^\circ\text{C} \rightarrow \text{linear}$

$$R_T = R_{T_0} (1 - \alpha_{T_0} (T - T_0))$$

T_0 : Ice point ~~temp~~ (0°C)



if $\Delta T > 50^\circ\text{C}$

$$\frac{R_{T_1}}{R_{T_2}} = e^{\beta \left(\frac{1}{T_1} - \frac{1}{T_2} \right)}$$

in K

$$R_T = A \left(\frac{R_{T_0}}{T_0} \right) e^{\beta/T}$$

in K
 at Ice point only

A, B: are constants
 $\beta = \beta \text{ (K)}$

Piezoelectric

→ Crystal → Quartz } natural
 → Rochelle salts }

→ P, F → Voltage

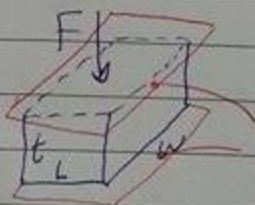
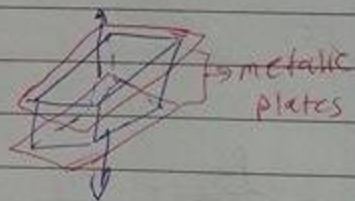
Synthetic Crystal (solid)

BT: Barium Titanate

LS: Lithium sulphate

(L W t)

Dimension for plate = Dimension for crystal



$$t_f = t_i - \Delta t$$

$$\text{charge } Q = K_c \cdot F$$

charge sensitivity

$$(C) = \left(\frac{C}{N} \right) (N)$$

— (1)

$$F = \frac{EA}{t} \Delta t \quad (2)$$

$$V_o = \frac{Q}{C} \quad (3)$$

$$C = \epsilon_0 \epsilon_r \frac{A}{t} \quad (4)$$

$$V_o = \left(\frac{K_c}{\epsilon_0 \epsilon_r} \right) (P t)$$

pressure

Crystal voltage sensitivity (K_{VC})

$$V_o = K_{VC} P t$$

$$= K_{VC} \frac{F}{A} t$$

$$V_o = K_{VC} \frac{E}{t} \Delta t t$$

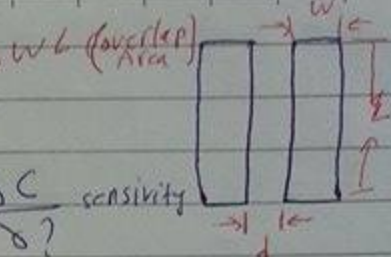
$$= K_{VC} E \Delta t$$

$$K_s = \frac{V_o}{\Delta t} = K_{VC} E$$

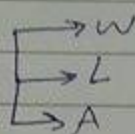
	K_{VC}	$\epsilon_0 \epsilon_r$	K_c ^{10^{-12}} pC/N
BT	12×10^{-3}	12.5×10^{-9}	150 ✓
Quartz Quartz	50×10^{-3} ✓	40.6×10^{-12} x	2 x

Capacitor

$$C = \epsilon \epsilon_0 \frac{A}{d}$$



modes of operations



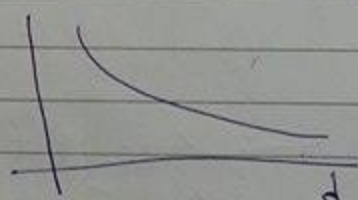
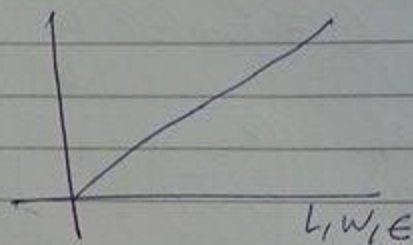
$\frac{\partial C}{\partial ?}$ sensitivity

$$\rightarrow w \rightarrow \frac{\partial C}{\partial w} = \frac{\epsilon \epsilon_0 L}{d}$$

$$\rightarrow L \rightarrow \frac{\partial C}{\partial L} = \frac{\epsilon \epsilon_0 w}{d}$$

$$\rightarrow d \rightarrow \frac{\partial C}{\partial d} = -\frac{\epsilon \epsilon_0 A}{d^2}$$

$$\rightarrow \epsilon \rightarrow \frac{\partial C}{\partial \epsilon} = \frac{A}{d}$$

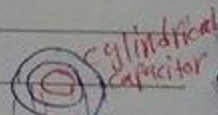


$$\frac{\Delta C}{C} \text{ \% out}$$

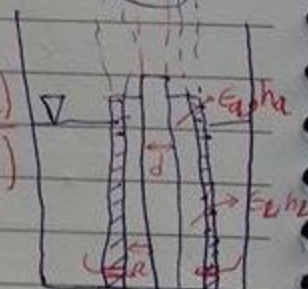
$$\frac{\Delta X}{X} \text{ \% input}$$

$$= K_c \quad (\text{capacitance factor})$$

~~operation~~ operation sensitivity



$$C = \frac{2\pi\epsilon_0\epsilon_r(h_1\epsilon_1 + h_2\epsilon_2)}{2\ln(1 - \frac{a}{r})}$$

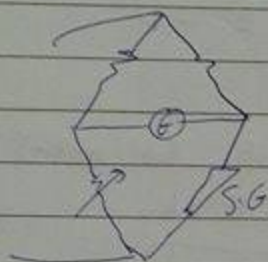


Strain gauge :-

it measures stress $\rightarrow \Delta R$
strain $\rightarrow \Delta R$

$$R = \frac{\rho L}{A} \quad , \quad \rho : \text{Resistivity } \Omega\text{-m}$$

$$\text{strain} = \frac{\Delta L}{L}$$



Wheatstone Bridge

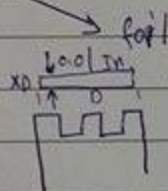
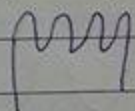
Strain gauges

metals

wire

$$A = \pi D^2$$

$$D = 10-25 \mu\text{m}$$



$$A = \pi D^2$$

semiconductor
tension, ΔR
compression, ΔR (wire case)

$$\text{max. for metals } \frac{\Delta R}{R} = \frac{\Delta L}{L} = \epsilon$$

$$\text{max. for semiconductor } \frac{\Delta R}{R} = -135 \epsilon$$

→ 135% tensile

$$R = \frac{\rho L}{A}$$

$$\ln R = \ln \rho + \ln L - \ln A$$

$$\frac{dR}{R} = \frac{d\rho}{\rho} + \frac{dL}{L} - \frac{dA}{A}$$

$$\frac{dA}{A} = \frac{d(\pi D^2)}{A} = 2 \frac{dD}{D}$$

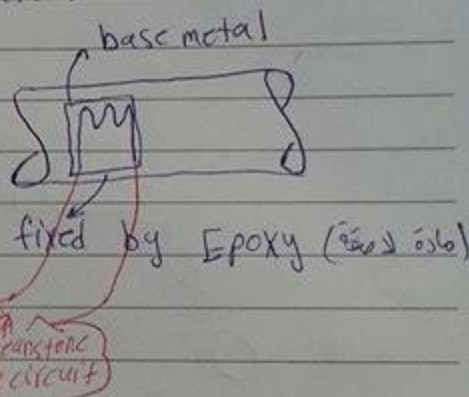
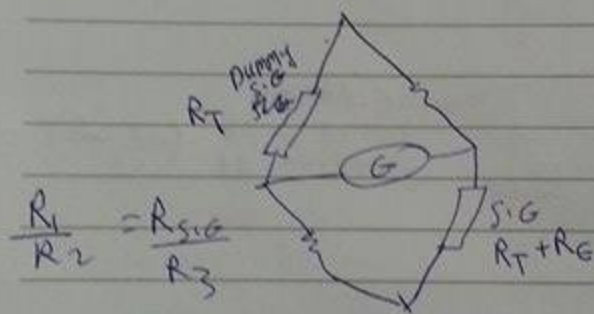
$$\frac{\Delta R}{R} = \frac{\Delta S}{S} + \frac{\Delta L}{L} - 2 \frac{\Delta D}{D}$$

(G_f) strain Gauge factor $\frac{\Delta R}{R} = G_f$

$$\frac{\Delta R}{R} = \frac{\Delta S/S}{\frac{\Delta L}{L}} + 1 + 2 \nu \quad \text{or very small} \quad \frac{\Delta D/D}{\frac{\Delta L}{L}} = -\nu$$

$G_f = 1 + 2\nu$ max $G_f = 2$, max $\nu = 0.5$ for metal

to eliminat the effect of temperature



Exam question 2

$$M_p = \frac{60-40}{40} = 0.5$$

$$= \frac{\pi \omega}{2\sqrt{1-\eta^2}}$$

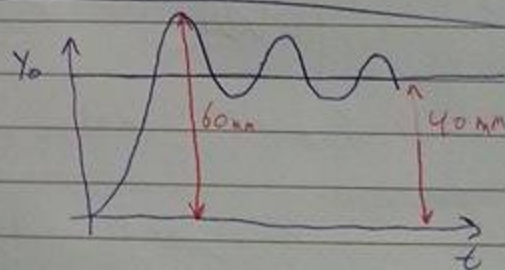
$$\eta = 0.2154$$

$$\omega_d = \frac{\pi}{t_p} \Rightarrow \omega_d = 7.618 \text{ rad/s}$$

$$\omega_n = \frac{\omega_d}{\sqrt{1-\eta^2}} = 7.681 \text{ rad/s}$$

$$\phi = \cos^{-1}(\eta) = 77.55^\circ = 1.3603 \text{ rad}$$

$$t_r = \frac{\pi - \phi}{\omega_n} = 0.683$$



$$\eta = \frac{B}{2.8 \sqrt{2agV}}$$

$$\omega_n = \sqrt{\frac{2ag}{V}}$$

$$\frac{\eta_1}{\eta_2} = \frac{\sqrt{V_2}}{\sqrt{V_1}} = \sqrt{\frac{1}{2}}$$

$$\eta_2 = 0.3047$$

$$\omega_{n2} = 3.8 \text{ rad/s}$$

$$\omega_{d2} = 3.61 \text{ rad/s}$$

$$t_{r2} = 0.57 \text{ sec}$$

Q2) First order because its controlling parameter is τ

$$K_s = 3 \text{ mm/c} = \frac{\alpha V_b}{A_c}$$

$$V_b = 123.75 \text{ mm}^3$$

$$\tau = \frac{8cV_b}{\pi A_b} = 34.135 \text{ sec}$$

for cylindrical bulb

$$d = 4.75 \text{ mm}$$

$$\tau_{cyl} = 39.3 \text{ sec}$$

$$X_o = K_s T_i \left(1 - e^{-t/\tau} \right) + K_s T_{init} \frac{t}{\tau}$$

$$X_o = 151.183 \text{ mm}$$

$$T_o = 50.4^\circ\text{C}$$

Q3) $\bar{T} = 12.06^\circ\text{C}$

$$\bar{D} = 0.752^\circ\text{C}$$

$$\sigma = 1.0084$$

$$\eta = 0.6801$$

$$V = \sigma^2 = 1.0169^\circ\text{C}$$

$$h =$$

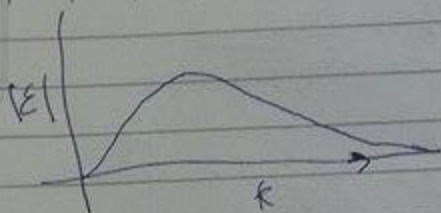
chapter (5): problems

Q2) $R_p = 2 \text{ k}\Omega$

$V_{in} = 10 \text{ V}$

$R_m = 5 \text{ k}\Omega$

$\frac{X_i}{X_t} = k = 0, 0.25, 0.5, 0.75, 1$



$$E = m V_i \left[\frac{k^2 (k-1)}{k(1-k) + \frac{R_m}{R_p}} \right] \quad E=0 \text{ at } k=0, 1$$

$$E(k=0.5) = m(10) \left[\frac{0.5^2 (0.5-1)}{0.5(1-0.5) + \frac{5}{2}} \right] = -0.454$$

Q3) $Q = K_c \frac{F}{\frac{C}{N}}$

$$Q = K_c P$$

$80 \times 10^{-12} \frac{C}{bar} \quad bar$

$C = 1 \times 10^{-9} F$

$Q = 80 \times 10^{-12} (1.4) = 112 \text{ pC}$

$$V_0 = \frac{Q}{C} = \frac{112 \times 10^{-12}}{1 \times 10^{-9}} = 112 \text{ mV}$$

Q4) $t = 2 \text{ mm}$

$K_{VC} = 0.055 \text{ V-m/N}$

$P = 1.5 \text{ MPa}$

$E = 40.6 \times 10^{-12} \text{ F/m}$

find V_0, K_c ?

$V_0 = K_{VC} t P = 165 \text{ V}$

$K_{VC} = \frac{K_c}{E} \Rightarrow K_c = 2.23 \text{ pC/N}$

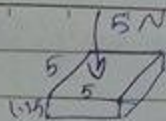
Q5)

$$k_c = 150 \text{ pc/N}$$

$$E = 12.5 \times 10^9 \text{ F/m}$$

$$E = 12 \times 10^6 \text{ N/m}^2$$

find strain, Q , C ?



$$\text{strain} = \frac{\text{stress} (F/A)}{E} = \frac{5/5 \times 5 \times 10^6}{12 \times 18} = 0.0167$$

$$Q = k_c F$$

$$= 150 \times 10^{-12} \times 5 = 750 \text{ pC}$$

$$C = \frac{Q}{V_0} = \frac{Q}{k_c \cdot t \cdot P} = \frac{750}{3} = 250 \text{ pF}$$

$\frac{k_c}{E}$ $\rightarrow F/A$

Q10) $T_1 = 100$, $T_2 = 130$, $T_0 = \frac{T_1 + T_2}{2} = 115$

$$R_{T_1} = 573.4, R_{T_2} = 606.5, R_{T_0} = 589.5$$

$$R_T = R_{T_0} (1 + \alpha_{T_0} (T - T_0))$$

$$\alpha_{T_0} = \frac{1}{R_{T_0}} \left(\frac{R_{T_2} - R_{T_1}}{T_2 - T_1} \right) = 0.00182 \quad (\text{دالة خطية})$$

linear

$$R_T = 589.5 (1 + 0.00182 (T - 115))$$

معادلة الخط الواحد جيد T_2, T_1

quadratic

$$R_{T_1} = R_{T_0} (1 + \alpha_1 (T_1 - T_0) + \alpha_2 (T_1 - T_0)^2)$$

Range (II)

$$R_{T_2} = R_{T_0} (1 + \alpha_2 (T_2 - T_0) + \alpha_2 (T_2 - T_0)^2)$$

14

$$\alpha_1 = 0.00159 \quad (\text{دالة خطية})$$

$$\alpha_2 = -15.098 \times 10^6 \quad (\text{معادلة تربيعية})$$

Five Apple

RTD $\alpha = +ve$
thermistor $\alpha = -ve$

Q11) $\alpha_{T_0} = 0.005392 \text{ } ^\circ\text{C}^{-1}$

$T_0 = 25^\circ\text{C} \rightarrow R_0 = 90 \text{ } \Omega$ (lowest)

$R_T = 115 \text{ } \Omega \rightarrow T ??$

$$R_T = R_{T_0} (1 + \alpha_{T_0} (T - T_0))$$

$$115 = 90 (1 + 0.005392 (T - 25)) \Rightarrow T = 76.51^\circ\text{C}$$

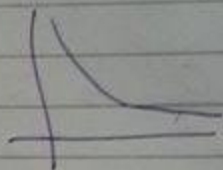
Q12) thermistor

$$\alpha_{T_0} = -0.05$$

25 - 50 assume linear

$R_{T_0} = 100 \text{ } \Omega \rightarrow T_0 = 25^\circ\text{C}$

$T = 35^\circ\text{C} \rightarrow R_T ??$



$$R_T = R_{T_0} [1 + \alpha (T - T_0)]$$

$$= 100 (1 - 0.05 (35 - 25))$$

$$R_T = 50 \text{ } \Omega$$

Q13) $R_{T_0} = 3980 \text{ } \Omega$, $T_0 = 0^\circ\text{C}$

$R_T = 794 \text{ } \Omega$, $T = 50^\circ\text{C}$

border case so logarithmic

$$R_T = A R_{T_0} e^{B/T}$$

logarithmic valid (Range 10 to 1000 K)

At $T = 0 = 273 \text{ K}$

$$R_T = R_{T_0} A e^{B/T}$$

$$3980 = A 3980 e^{B/273}$$

$$\Rightarrow 1 = A e^{B/273}$$

— (1)

at $T = 50 = 323 \text{ K}$

$$794 = A 3980 e^{B/323} \Rightarrow \frac{794}{3980} = A e^{B/323}$$

$$A = 30 \times 10^{-6}$$

$$B = 7845 \text{ K}$$

OR

$$T_1 = 0^\circ \text{C}$$

$$R_{T_1} = 3980 \Omega$$

$$T_2 = 50^\circ \text{C}$$

$$R_{T_2} = 794 \Omega$$

$$R_{T_2} = R_{T_1} e^{B(\frac{1}{T_2} - \frac{1}{T_1})}$$

$$B = 2845 \text{ K}$$

then use any Temp to find A

Q14)

$$C_1 = \epsilon_0 \epsilon_r \frac{A}{d_1}$$

$$= 22.125 \text{ pF}$$

$$d_2 = 0.18 \text{ mm}$$

$$C_2 = \epsilon_0 \epsilon_r \frac{A}{d_2}$$

$$= 24.583 \text{ pF}$$

ratio per unit change of capacitance
to per unit change in displacement = K_c

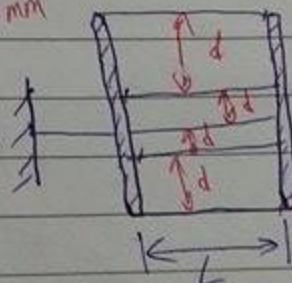
$$K_c = \frac{\frac{\Delta C}{C_1}}{\frac{\Delta d}{d_1}} = 1.11$$

Q14)

$$C = \frac{N \epsilon_0 \epsilon_r (L \times W)}{d}$$

$$C = \frac{N \epsilon_0 \epsilon_r (L-x) W}{d}$$

$$d = 0.25 \text{ mm}$$



$$\frac{\partial C}{\partial x} = - \frac{N \epsilon_0 \epsilon_r W}{d} = -3540 \text{ pF/m}$$

strain Gauge examples

EX1: wire S.G, $G_f = 2$

$$\sigma = 100 \text{ MPa}$$

$$E = 200 \text{ GPa}$$

$$\frac{\Delta R}{R} = G_f \epsilon$$

$$\frac{\Delta R}{R} = 2 \frac{100 \times 10^6}{200 \times 10^9} = 0.1\%$$

Rosette: multiple strain Gauge to measure one input at series

EX2:- beam $\rightarrow L = 0.1 \text{ m}$
 $\rightarrow A_c = 3 \text{ m}^2$
 $\rightarrow E = 207 \text{ GPa}$

unstrained Resistance $R = 240 \Omega$

$$G_f = 2.2$$

$$\Delta R = 0.013 \text{ (tensile force)}$$

$$G_f = \frac{\Delta R}{R} \cdot \frac{L}{\Delta L} \Rightarrow \Delta L = 2.46 \times 10^{-6} \text{ m}$$

$$\epsilon = \frac{\Delta R}{R} = G_f \epsilon$$

$$\epsilon = \frac{\Delta L}{L} = \frac{\sigma}{E}$$

$$\sigma = 5.092 \times 10^6 \text{ N/m}^2$$

$$\sigma = F/A$$

$$F = 2.037 \times 10^3 \text{ N}$$

$$F = \frac{5.092 \times 10^6}{3}$$

$$= 1697333.333$$

EX3:- $G_f = 1.2$

↳ not semiconductor (composite material)
tensile force

$$G_f = 1 + 2V + \frac{\Delta R}{R} \rightarrow 0$$

$$\frac{1.2 - 1}{2} = V \Rightarrow V = 0.1$$

EX4) $\epsilon = 5 \mu$

$G_f = -12.1$

Nichrome

$R = 120 \Omega$

(designed for compressive loading)

(designed for tensile loading)

$$R_t = \epsilon G_f R_i + R_i$$

compressive force ϵ negative

for Ni

$$R_t = -5 \times 10^{-6} \times -12.1 \times 120 + 120$$

$$= 120 + 7.26 \times 10^{-3}$$

$\Delta R = 7.26 \text{ m}\Omega$

for Nichrome

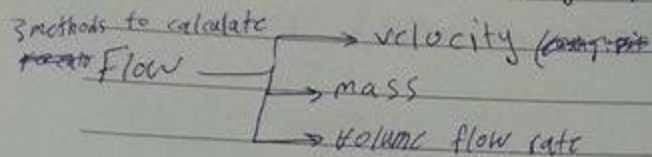
$$R_t = -5 \times 10^{-6} \times 2 \times 120 + 120$$

$$R_t = 120 - 1.2 \times 10^{-3}$$

$$\Delta R = -1.2 \times 10^{-3} \text{ m}\Omega$$

Flow, pressure and velocity measurements :-

3 methods to calculate



Methods → Head type $\dot{V} = f(\Delta P)$ (orifice, venturi)

→ variable head $\dot{V} = f(h)$ (Rotameter)

→ turbine meter (~~inlet outlet~~)

→ magnetic flow meters $\dot{V} = f(\phi)$ flux

→ Thermal flow meters (Hot wire Anemometer) $\dot{V} = f(\Delta T)$

→ positive displacement (reciprocated pumps) $\dot{V} = f(\Delta P)$

→ mass method $\dot{m} = f(\dot{V})$

Flow → laminar, Turbulent

$Re \leq 2000$ Laminar

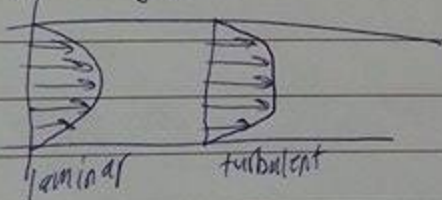
Flow → compressible, Incompressible $\Delta \dot{V} = f(P)$

compressibility factor γ

$\gamma = 1$ liquid (Incompressible)

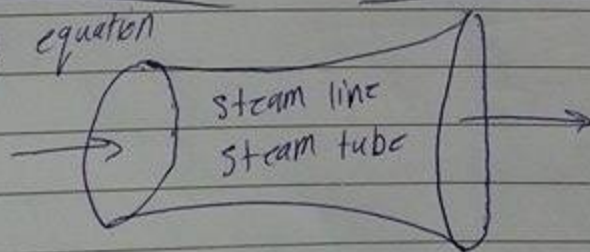
$\gamma < 1$ compressible

velocity profile



continuity equation

$$\sum \dot{m}_{in} = \sum \dot{m}_{out}$$



steam tube: tube with no leakage.
(no cross flow)

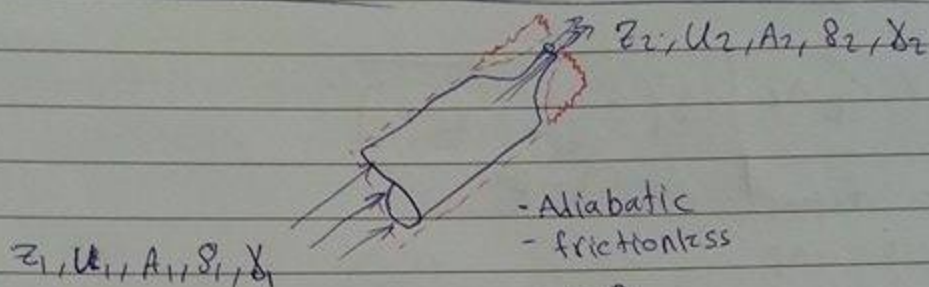
Bernoulli theorem:

$$\dot{Q} \propto (P_1 - P_2)^n$$

$\rightarrow$ 1 laminar
 $\rightarrow$ 5/4 turbulent

$\dot{Q}$: volumetric flow rate

Obstruction flow measurements devices (Head type)



- Adiabatic
- frictionless
- 1-D
- Irrotational
- Isothermal
- Stream Tube
- Incompressible

$$\rho_1 A_1 u_1 = \rho_2 A_2 u_2$$

$$A_1 u_1 = A_2 u_2 \quad (1)$$

$$\frac{P_1}{\rho_f} + \frac{u_1^2}{2g} + z_1 = \frac{P_2}{\rho_f} + \frac{u_2^2}{2g} + z_2$$

$$\left[\frac{P_1 - P_2}{\rho_f} + z_1 - z_2 + \frac{u_1^2}{2g} \right] * 2g = u_2^2$$

$$u_{2th} = \sqrt{2g \left[\frac{P_1 - P_2}{\rho_f} + \underbrace{z_1 - z_2}_{\text{negative}} \right] / \left(1 - \left(\frac{A_2}{A_1} \right)^2 \right)} \quad \text{m/s}$$

$$\left(\frac{d_2}{d_1} \right)^4 = \beta^4$$

$$\dot{Q}_{th} = A_2 u_{2th} \quad \text{m}^3/\text{s}$$

$$\dot{m}_{th} = \rho_2 A_2 u_{2th}$$

$$C_D = \frac{\dot{Q}_{act}}{\dot{Q}_{th}} \quad (\text{vena contracta})$$

$$\dot{Q}_{act} = C_D A_2 U_{2th}$$

ΔP_{12} in ~~static~~ pa

$$\dot{Q}_{act} = \gamma C_D A_2 \sqrt{2g \left(\frac{\Delta P_{12}}{\gamma_f} + z_1 - z_2 \right)}$$

$\gamma \rightarrow 1$ incomp
 $\gamma < 1$ comp

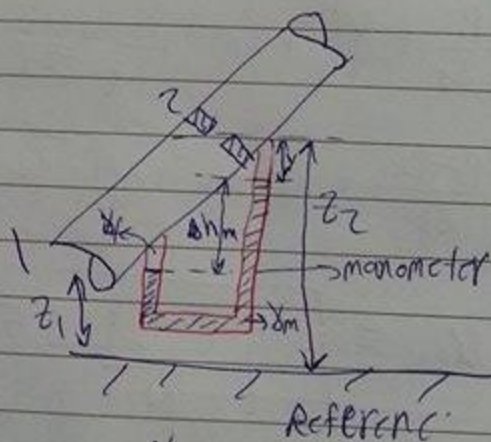
$$\dot{m}_{act} = \rho_2 \dot{Q}_{act} \rightarrow \begin{cases} \rho_2 = \rho_1 \text{ liquid} \\ \frac{\rho_1}{\rho_2} = \left(\frac{\rho_1}{\rho_2} \right)^{1/K} \text{ (gas)} \end{cases}$$

$$\rho_2 = \left(\frac{\rho_2}{\rho_1} \right)^{1/K} \rho_1$$

$$P_1 + \gamma_f x = P_2 + \gamma_f y + \gamma_m h$$

$$P_1 - P_2 = \gamma_f (y - x) + \gamma_m h$$

$$\frac{P_1 - P_2}{\gamma_f} + (z_1 - z_2) = h_m \left(\frac{\gamma_m}{\gamma_f} - 1 \right)$$



$$\dot{Q}_{act} = \frac{\gamma_f C_D A_2}{\sqrt{1 - \beta^4}} \sqrt{2g h_m \left[\frac{\gamma_m}{\gamma_f} - 1 \right]}$$

γ_m : manometer
 γ_f : fluid

$$\frac{\gamma_m}{\gamma_f} = \frac{\rho_m}{\rho_f} = \frac{\rho_m G_m}{\rho_f G_f}$$

$$Re_{d_2} = \frac{\rho_2 U_{act} d_2}{\mu_f}$$

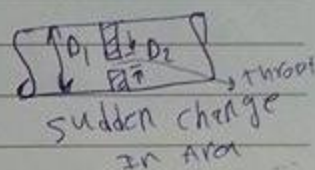
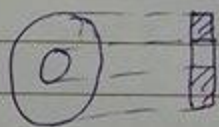
$$Q_{act} = C_D A_2 U_{2th}$$

$$U_{act} = C_D U_{2th}$$

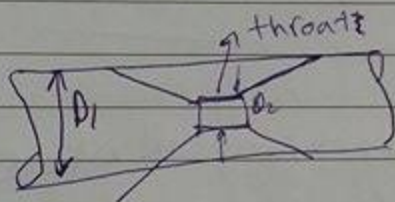
compressible $\rightarrow$ mass flow rate

Incompressible $\rightarrow$ volume flow rate

Orifice plate

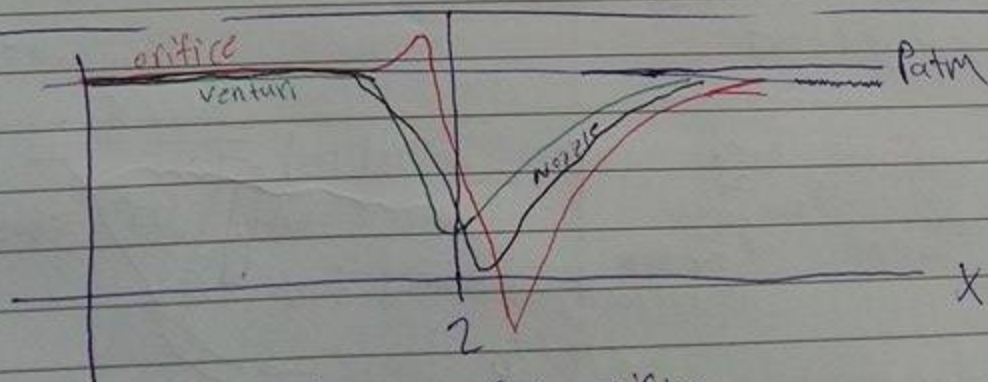
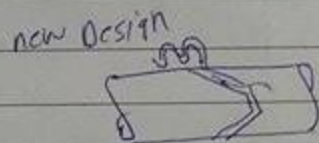
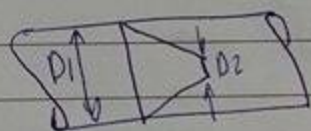


Venturimeter



venturi is better because Area changes gradually.

flow nozzle



$\Delta P_{loss \max}$ for orifice

$\Delta P_{loss \min}$ for venturi

ΔP has an effect if there is a steady flow device after the obstruction device.

to eliminate that effect

1- ~~put~~ put the steady flow device (pump) ^{far} away from the obstruction device

2- use compensating pump to increase the inlet pressure

$$W_p = C_{fact} \times \frac{\Delta P_{loss} \rightarrow Pa}{\rho_{pump}} \quad (W)$$

$$cost = W_p \times \text{tariff rate} \times \text{time}$$

$$kW \times \frac{JD}{kWh} \times \frac{hrs}{Year}$$

selection criteria :-

- 1) space
- 2) cost
- 3) Accuracy
- 4) ΔP_{loss} tolerated
- 5) nature of flow

accuracy depends on

- 1) β : $\frac{A_{tube}}{A_{orifice}}$ Area ratio
- 2) C_D : $\frac{C_{D,tube}}{C_{D,orifice}}$ C_D ratio
- 3) $\rho(T)$ density of fluid
- 4) Temperature effect

Rotameter variable head (act as control too)

Area not constant

the height of the Bob is not constant

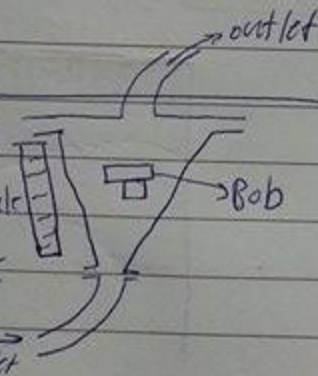
Buoyancy force

Scale

Bob

Inlet

$$F_D + F_B = F_W$$



frontal area of the Bob
(Projected Area)

$$\frac{C_D A_b S_f U_m^2}{2} + S_f V_b g = S_b V_b g$$

controlling parameter

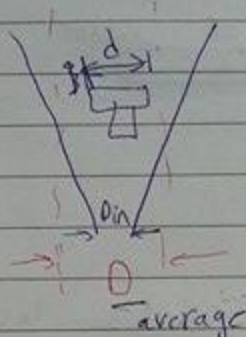
$$U_m = \sqrt{\frac{2(S_b - S_f) V_b g}{C_D A_b S_f}}$$

$$= \sqrt{\frac{2g}{C_D} \left(\frac{S_b}{S_f} - 1 \right) \left(\frac{V_b}{A_b} \right)}$$

$$Q = U_m \times A$$

$$A = \frac{\pi}{4} \left[(D_{in} + ay)^2 - d^2 \right]$$

$$= \frac{\pi}{4} \left[(D_{out} - ay)^2 - d^2 \right]$$



D_{in} D_{out} y d

y : height of the float

optimum if
when $S_b = 2S_f$

disadvantages

- 1- glass
- 2- ~~not~~ dirt ~~the~~ existence in water

velocity → pitot Tube

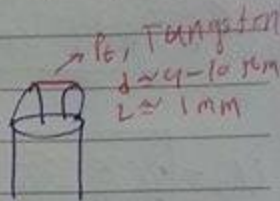
→ Hot wire Anemometer
metal

$$R = \frac{8L}{A}, R = f(T)$$

based on: forced convection

has length l & cross sectional area A_c length l goes to h
 A_c goes to h also, has loose l // // // //

cross sectional area of the flow



wire
 molten glass
 1- to protect it from dirt

2- to protect it from thermal shock (resistance)

OR tube

$$I^2 R_w = h A_w (T_w - T_f)$$

$$R_w = R_{ref} [1 + \alpha (T_w - T_{ref})]$$

King's law

$$h = a + b U_f^c \quad ; a, b, c : \text{properties of flow}$$

$$U_f = \left\{ \frac{I^2 R_{ref} [1 + \alpha (T_w - T_{ref})] - a}{b A_w (T_w - T_f)} \right\}^{1/c}$$

Process

1- constant Temp (T)
 variable current (I)

R_w : constant

convection

$$U_f = f(I, T_f)$$

T_w : wire temp

2- constant current (I)

variable Temp T

R_w : variable

$$U_f = f(T_w, T_f)$$

Examples:-

$$\mu_{\text{water}} = 1.08 \times 10^{-3} \quad (\text{initial})$$

$$S_w = 1000$$

$$D_1 = 400 \text{ mm}$$

$$d_2 = 150 \text{ mm}$$

$$h_m = 250 \text{ mm of mercury}$$

$$S.G. \text{ Hg} = 13.6$$

$$C_p = 0.98$$

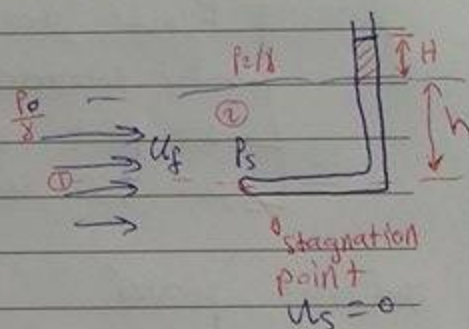
$$\Delta P_{1-2} = \rho g h \quad (\text{single homogenous fluid})$$

$$Q = \frac{C_p A_2}{\sqrt{1 - \beta^4}} \sqrt{2g h_m \left[\frac{S.G_m}{S.G_f} - 1 \right]}$$

Pitot tube $\rightarrow$ velocity

$\Delta KE_{\text{flow}} \rightarrow$ pressure E

$$\frac{P_0}{\gamma} + \frac{U_0^2}{2g} + Z_1 = \frac{P_s}{\gamma} + \frac{U_s^2}{2g} + Z_s$$



$$\frac{U_0^2}{2g} = \frac{P_s - P_0}{\gamma}$$

$$U_0 = \sqrt{\frac{2}{\gamma} (P_s - P_0)}$$

$$P_0 = \gamma h = P_s$$

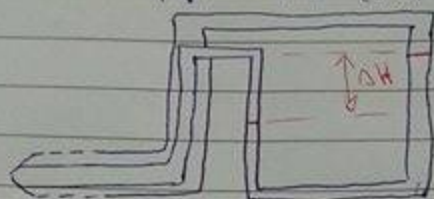
$$P_s = \gamma (h + H)$$

$$P_s - P_0 = \gamma H$$

$$U_{0_{th}} = \sqrt{\frac{2}{\gamma} \gamma H} = \sqrt{2gH}$$

$$U_{oact} = C U_{oth}$$

↳ flow coeff.
pitot tube coeff



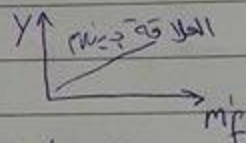
used when
1- Gas flow
2- High speed

pitot tube → can dust fall

$$U_{oact} = C_s \sqrt{2g H_m \left[\frac{S.G_m}{S.G_f} - 1 \right]}$$

calibration process

- to find discharge coeff. for obstruction
- Flow measurement devices
- for Rotameter, find slope a
- for pitot to find C
- Hot wire Anemometer, to find a, b, C



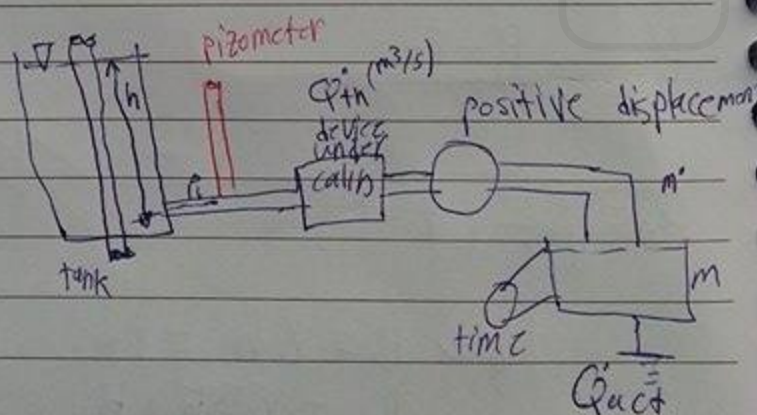
*flow prover

theory principle: catch and weigh

density ρ error $\Delta \rho$, ρ is not

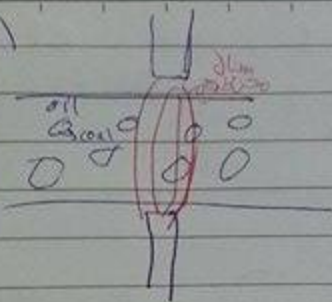
P_1 must be const

$$P_1 = \rho g h$$



magnetic flow meter (dab sic)

slurry
flow
(Liquid + solids)



Ex:- 1)

$$\gamma_{\text{water}} = 9810 \frac{\text{N}}{\text{m}^3}$$

$$\beta = \frac{150}{400} = 0.375$$

$$\gamma = 1 \quad (\text{water})$$

$$z_1 - z_2 = 0$$

$$h_m = 250 \text{ mm Hg}$$

$$Q' = \frac{C_D \gamma A_2}{\sqrt{1-\beta^4}} \sqrt{2g h_m \left[\frac{\gamma_m}{\gamma} - 1 \right]}$$

$$= \frac{(0.98)(1) \left(\frac{\pi}{4} \left(\frac{150}{1000} \right)^2 \right)}{\sqrt{1-(0.375)^4}} \sqrt{2(9.81) \left(\frac{250}{1000} \right) \left[\frac{13.6}{1} - 1 \right]}$$

$$= 0.1376 \text{ m}^3/\text{s}$$

or

$$Q' = \frac{C_D \gamma A_2}{\sqrt{1-\beta^4}} \sqrt{2g \left(\frac{\Delta P_{1-2}}{\gamma_f} + z_1 - z_2 \right)}$$

$$(z_1 - z_2) + \frac{\Delta P_{1-2}}{\gamma_f} = h_m \left[\frac{13.6}{1} - 1 \right] = 3.15 \text{ m}$$

Five Appl

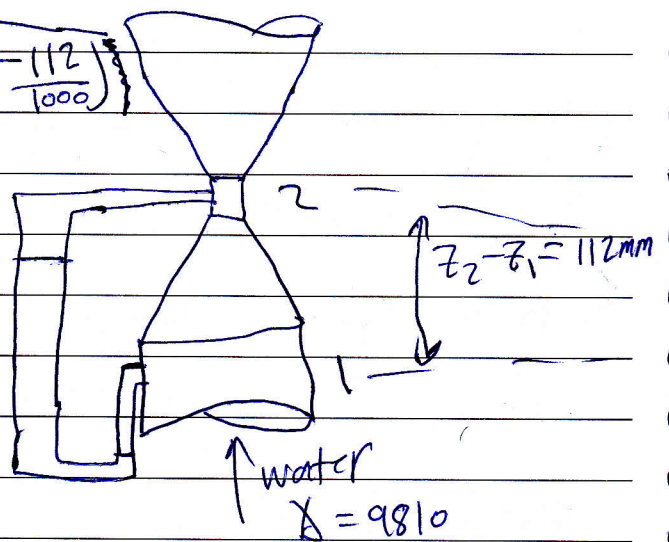
if said $\Delta P_{1-2} = 280 \text{ kPa}$
 ΔP_{1-2} given in bar ~~13~~ 300
 in Pa

EX2

$$Q = \frac{1 + 0.97 \times \frac{\pi}{4} \left(\frac{80}{1000}\right)^2}{\sqrt{1 - \left(\frac{80}{100}\right)^4}} \sqrt{2 \times 9.81 \left(\frac{30.6 \times 10^3}{9810} - \frac{112}{1000}\right)}$$

$$U_{2, \text{act}} = \frac{Q}{\frac{\pi}{4} \left(\frac{80}{1000}\right)^2} = 9.69 \text{ m/s}$$

$$Re_{d2} = \frac{8 U_{2, \text{act}} d_2}{\mu}$$



$$\left[\frac{30.6 \times 10^3}{9810} - \frac{112}{1000}\right] = h_m [13.6 - 1]$$

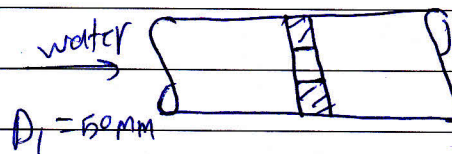
EX3:

$$C_D = 0.61, \quad Re = 1000$$

find $U_{2, \text{act}}$, ΔP_{1-2}

ν_f : kinematic viscosity = 1×10^{-6}

assume horizontal



$$d_1 = 60 \text{ mm}$$

$$d_2 = 30 \text{ mm}$$

$$\beta = 0.6$$

$$Re_{d2} = 100000$$

$$Re_{d2} = \frac{U_{2, \text{act}} d_2}{\nu_f}$$

$$100000 = \frac{U_{2, \text{act}} (0.03)}{1 \times 10^{-6}} \Rightarrow U_{2, \text{act}} = 3.33 \text{ m/s}$$

$$U_{2, \text{actual}} = C_D \sqrt{\frac{2(9.81) \left(\frac{\Delta P_{1-2}}{1000 \times 9.81}\right)}{1 - (0.61)^4}} \Rightarrow \Delta P_{1-2} = 12992.71 \text{ Pa}$$

EX 4

Air, $D_1 = 60 \text{ mm}$

$$\beta = 0.4$$

$$h_m = 250 \text{ cm}$$

$$P_1 = 93.7 \text{ KPa (absolute)}, T_1 = 20^\circ \text{C}$$

$$C_D = 0.675, \gamma = 0.92$$

$$\rho_1 = \frac{P_1}{RT_1} = \frac{93.7}{(0.287)(293)} = 1.1135 \text{ kg/m}^3$$

$$\dot{Q} = \frac{C_D \gamma A_2}{\sqrt{1-\beta^4}} \sqrt{2gh_m \left[\frac{S.G_m}{S.G_f} - 1 \right]} = 0.0528 \text{ m}^3/\text{s}$$

$\rightarrow \frac{1.1135}{1000}$

$$\dot{m} = (1.1135)(0.0528) = 0.0588$$

EX 5

$$S.G_f = 1.6$$

$$C = 1$$

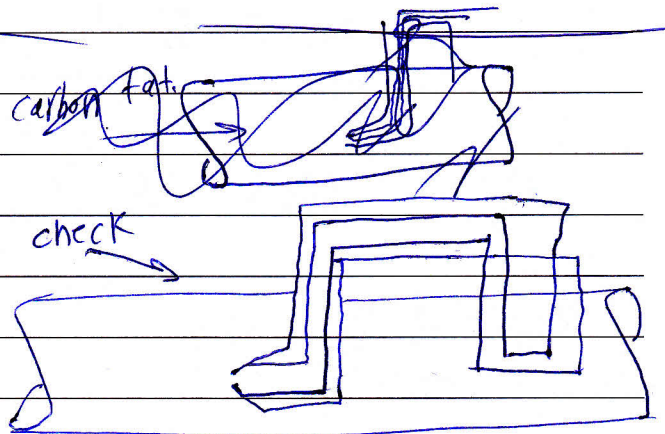
$$h_m = 8 \text{ cm Hg}$$

$$S.G_m = 13.6$$

$$U_{\text{fact}} = C \sqrt{2gh_m \left(\frac{S.G_m}{S.G_f} - 1 \right)}$$

$\frac{8}{100}$ $\frac{13.6}{1.6}$

$$= 3.43 \text{ m/s}$$



EX 6

$$U_{\text{fact}} = C \sqrt{2gh}$$

$$5 \text{ m/s} = 1 \sqrt{2gh}$$

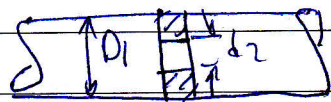
$$\Rightarrow h = 1.274 \text{ m}$$

$$h = \frac{P_s - P_o}{\gamma_f} = \frac{P_s - \gamma_f z}{\gamma_f} \Rightarrow P_s = 213.91 \text{ KPa}$$

Pitot tube application

سواء في
السرعة أو الارتفاع

EX: $D_1 = 20 \text{ cm}$
 $d_2 = 10 \text{ cm}$
 $h_m = 50 \text{ cm Hg}$
 $S.G_m = 13.5$



$$\dot{Q} = \frac{Y C_D A_2}{\sqrt{1 - \beta^4}} \sqrt{2g h_m \left[\frac{S.G_m}{S.G_f} - 1 \right]}$$

flow coeff (K_o)

$\frac{1}{\sqrt{1 - \beta^4}}$ velocity approach factor (E)

using Figure 10.6

$$\beta = 0.5$$

$$Re_{d_1} = \frac{S_f U_{act} d_1}{\mu_f}$$

$$\dot{Q}_{act} = A_1 U_{act} \Rightarrow U_{act} = \frac{\dot{Q}_{act}}{A_1}$$

$$Re_{d_1} = \frac{4 \dot{Q}_{act}}{\pi d_1 \mu_f}$$

find K_o when the curve becomes flat

$$K_o = 0.625$$

$$K_o = \frac{C_D Y}{\sqrt{1 - \beta^4}} \Rightarrow C_D =$$

$$\dot{Q} = 0.054 \text{ m}^3/\text{s}$$

now to verify the answer

$$\text{find } Re_{d_1} = \frac{(4)(0.054)}{\pi (0.2)(\mu_f)} = 3.2 \times 10^5$$

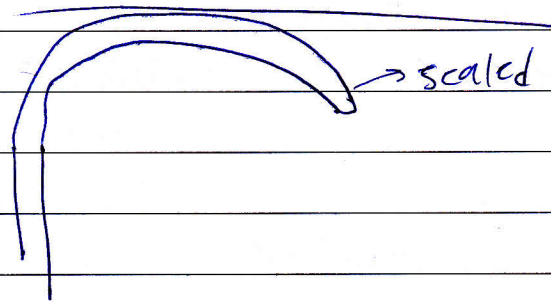
3.2×10^5 > point at which flat curve start

then C_D is true ~~if~~ ~~answer~~ ~~point~~ ~~at which~~ ~~Apple~~

If answer (Re_{d1}) < point at which flat curve start

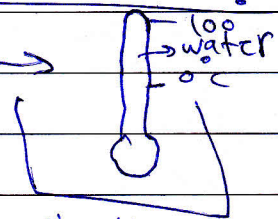
then use the value of CD ~~from~~ at Re_{d1} calculated and make iteration.

Pressure Measurement



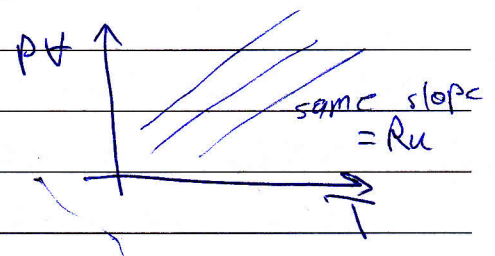
Temperature Measurement
not reproducible

using water
in thermometer



we use Gas thermometer for calibration $\rightarrow$ calibration liquids

$$\frac{PV}{T} = n R_u \quad \text{constant slope}$$



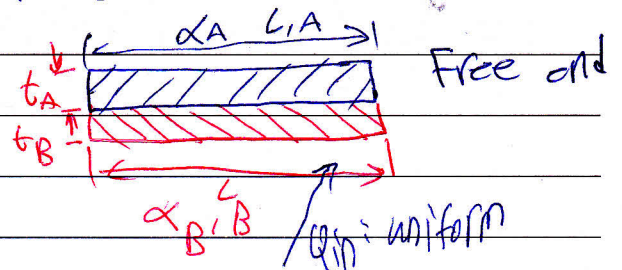
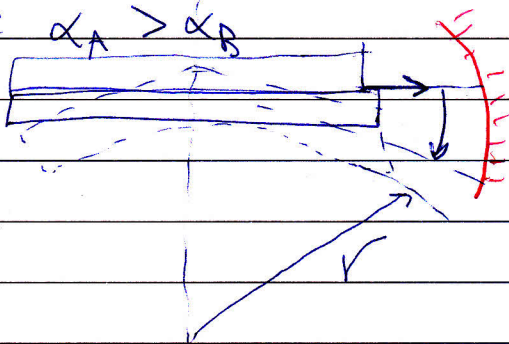
Bimetallic (thermostat) (temperature measurement and control)

Thermocouple

Piroumeter ($\text{مقياس التمدد الحراري} \text{ (الخطي)}$)

Bimetallic $\Rightarrow$ basic: Expansion of solids with ΔT

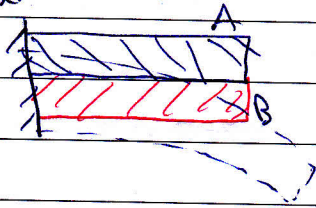
if $\alpha_A > \alpha_B$



Invar material : very small coeff of expansion
 $\alpha = 1.7 \times 10^{-6}$ (smallest)

Yellow Brass : $\alpha = 20.2 \times 10^{-6}$

Fixed end



for free end

$$r(\text{radius of curvature}) = \left(\frac{t}{\delta} \right) \left[\frac{3(1+m)^2 + (1+mn)(n^2 + \frac{1}{mn})}{(\alpha_H - \alpha_L)(T_2 - T_1)(1+m)^2} \right]$$

$$t = t_A + t_B$$

$$m = \frac{t_H \alpha_H}{t_L \alpha_L}, \quad t: \text{thickness}$$

$$n = \frac{E_H}{E_L}, \quad E: \text{modulus of elasticity}$$

T_2 = temp. to be measured

T_1 : Bonding temperature

~~for fixed end~~

if $t_A = t_B \rightarrow m = 1$

$E_H = E_L \rightarrow n = 1$

$$r = \frac{2t}{3(\alpha_H - \alpha_L)(T_2 - T_1)}$$

for fixed end (cantilever)

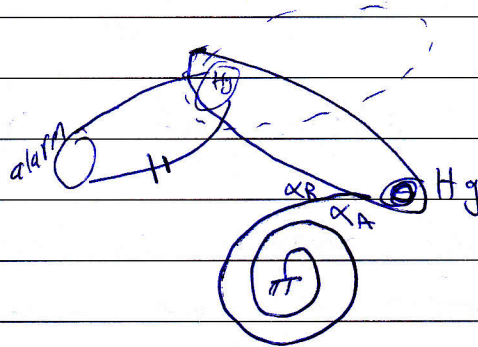
$$r = \frac{t}{2} \left[\frac{1 + \alpha_L(T_2 - T_1)}{(\alpha_H - \alpha_L)(T_2 - T_1)} \right]$$

Application

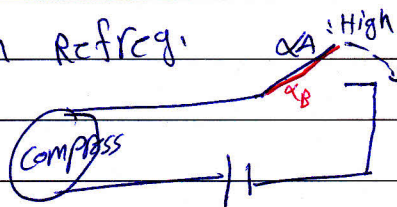
welding machine



2- fire alarm



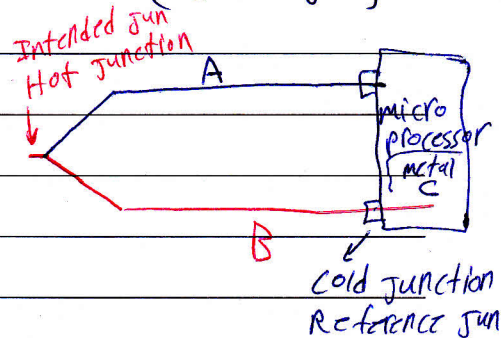
3- ~~therm~~ thermostat in Refrig.



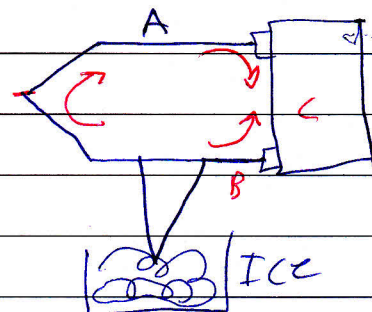
Thermocouple (different metals) (different temp.)

basic: seebeck ~~effect~~ effect theory: temp can be measured by converting it Electromotive force (voltage)

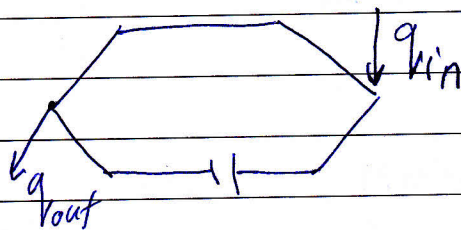
1- calibration: from tables get the voltage of AC and BC and subtract them from the processor reading



2-



Thermoelectric Refrig.



Seebeck voltage = Peltier emf + Thomson emf

Thomson : Voltage produced in case of different temperatures of the two ends of the wire

$$emf = \underbrace{\alpha}_{\text{voltage const}} \Delta T \rightarrow \text{temp}$$